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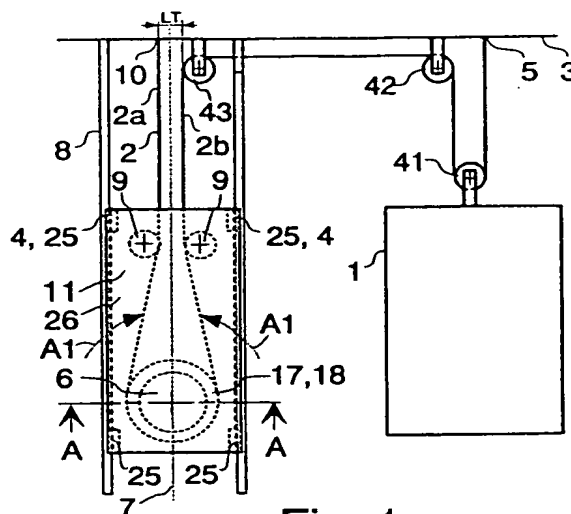
(11) Publication number:

**0 631 969 A2**

(12)

**EUROPEAN PATENT APPLICATION**(21) Application number: **94109886.5**(51) Int. Cl.<sup>6</sup>: **B66B 11/04**(22) Date of filing: **27.06.94**(30) Priority: **28.06.93 FI 932972**(43) Date of publication of application:  
**04.01.95 Bulletin 95/01**(84) Designated Contracting States:  
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**D-80639 München (DE)**(54) **Elevator drive machine placed in the counterweight.**

(57) In the present invention, a rotating elevator motor (6) provided with a traction sheave (18) is placed in the counterweight (26) of an elevator suspended with ropes (2). A gear system is not necessarily needed because, according to the invention, the structure and placement of the motor allow the use of a motor having a large diameter and a high torque. As the length of the motor still remains small, the motor/counterweight of the invention can be accommodated in the space normally reserved for a counterweight in the elevator shaft. The motor shaft lies in the counterweight (26) substantially midway between the guide rails (8) and the number (a,b) of ropes is equal on both sides of the plane (24) going through the centre lines of the guide rails.

**Fig. 1****EP 0 631 969 A2**

The present invention relates to the counterweight of a rope-suspended elevator moving along guide rails and to an elevator drive machinery/motor placed in the counterweight, said motor comprising a traction sheave, a bearing, an element supporting the bearing, a shaft, a stator provided with a winding, and a rotating rotor.

Traditionally, an elevator machinery consists of a hoisting motor which, via a gear, drives the traction sheaves around which the hoisting ropes of the elevator are passed. The hoisting motor, elevator gear and traction sheaves are generally placed in a machine room above the elevator shaft. They can also be placed beside or below the elevator shaft. Another known solution is to place the elevator machinery in the counterweight of the elevator. Previously known is also the use of a linear motor as the hoisting machine of an elevator and its placement in the counterweight.

Conventional elevator motors, e.g. cage induction, slip ring or d.c. motors, have the advantage that they are simple and that their characteristics and the associated technology have been developed during several decades and have reached a reliable level. In addition, they are advantageous in respect of price. A system with a traditional elevator machinery placed in the counterweight is presented e.g. in publication US 3101130. A drawback with the placement of the elevator motor in this solution is that it requires a large cross-sectional area of the elevator shaft.

Using a linear motor as the hoisting motor of an elevator involves problems because either the primary part or the secondary part of the motor has to be as long as the shaft. Therefore, linear motors are expensive to use as elevator motors. A linear motor for an elevator, placed in the counterweight, is presented e.g. in publication US 5062501. However, a linear motor placed in the counterweight has certain advantages, e.g. that no machine room is needed and that the motor requires but a relatively small cross-sectional area of the counterweight.

The motor of an elevator may also be of the external-rotor type, in which the traction sheave is joined directly to the rotor. Such a structure is presented e.g. in publication JP 5232870. The motor is gearless. The problem with this structure is that, to achieve a sufficient torque, the length and diameter of the motor have to be increased. In the structure presented in US 4771197, the length of the motor is further increased by the brake, which is placed alongside of the rope grooves. Moreover, the blocks supporting the motor shaft increase the motor length still further. If a motor according to US 4771197 is placed in the counterweight, the counterweight must have larger dimensions and cannot be accommodated in the space normally reserved

for a counterweight.

Another previously known elevator machine is one in which the rotor is inside the stator and the traction sheave is attached to a disc placed at the end of the shaft, forming a cup-like structure around the stator. Such a solution is presented in Fig. 4 in publication US 5018603. Fig. 8 in the same publication presents an elevator motor in which the air gap is oriented in a direction perpendicular to the motor shaft. Such a motor is called a disc motor or a disc rotor motor. These motors are gearless, which means that the motor is required to have a higher torque than a geared motor. The required higher torque again increases the diameter of the motor.

The object of the present invention is to produce a new structural solution for the placement of a rotating motor in the counterweight of an elevator, designed to eliminate the above-mentioned drawbacks of elevator motors constructed according to previously known technology.

The invention is characterized by what is presented in the characterization part of claim 1. Other embodiments of the invention are characterized by the features presented in claims 2-15.

The advantages of the invention include the following:

Placing the elevator motor in the counterweight as provided by the invention allows the use of a larger motor diameter without involving any drawbacks.

A further advantage is that the motor can be designed for operation at a low speed of rotation, thus rendering it less noisy. Having a high torque, the motor does not necessarily require a gear, although this could also be built inside the motor, e.g. a epicyclic gearing between rotor and traction sheave.

As compared with a linear motor, the motor of the invention provides the advantage that it makes it unnecessary to build an elevator machine room and a rotor or stator extending over the whole length of the elevator shaft.

The present invention also solves the space requirement problem which results from the increased motor diameter and which restricts the use of a motor according to publication US 4771197. Likewise, the length of the motor, i.e. the thickness of the counterweight is substantially smaller in the motor/counterweight of the invention than in a motor according to US 4771197.

A further advantage is that the invention involves a saving in counterweight material corresponding to the weight of the motor.

The motor/counterweight of the invention has a very small thickness dimension (in the direction of the motor shaft), so the cross-sectional area of the motor/counterweight of the invention in the cross-section of the elevator shaft is also small and the

motor/counterweight can thus be easily accommodated in the space normally reserved for a counterweight.

According to the invention, the placement of the motor in the counterweight is symmetrical in relation to the elevator guide rails, which is an advantage regarding the guide rail strength required.

The motor may be a cage induction, reluctance or asynchronous motor.

In the following, the invention is described in detail in the light of an embodiment by referring to the drawings, in which

Fig. 1 presents a diagrammatic illustration of an elevator motor according to the invention, placed in the counterweight and connected to the elevator car by means of ropes.

Fig. 2 presents a cross-section of an elevator motor placed in the counterweight according to an embodiment of the invention, and

Fig. 3 presents a cross-section of an elevator motor placed in the counterweight according to another embodiment of the invention.

In Fig. 1, the elevator car 1, suspended on ropes 2, moves in the elevator shaft in a substantially vertical direction. One end of each rope is anchored at point 5 at the top part 3 of the shaft, from where the ropes are passed over a diverting pulley 41 on the elevator car 1 and diverting pulleys 42 and 43 at the top part 3 of the shaft to the traction sheave 18 of the elevator motor 6 in the counterweight 26 and further back to the shaft top, where the other end of each rope is anchored at point 10. The counterweight 26 and the elevator motor 6 are integrated in a single assembly. The motor is placed substantially inside the counterweight, and the motor/counterweight moves vertically between the guide rails 8, which receive the forces generated by the motor torque. The counterweight 26 is provided with safety gears 4 which stop the motion of the counterweight in relation to the guide rails 8 when activated by an overspeed of the counterweight or in response to separate control. The space LT required by the rope sets in the horizontal direction of the shaft is determined by the diverting pulleys 9 on the counterweight, the point 10 of rope anchorage and the position of diverting pulley 43 at the shaft top 3. By suitably placing the diverting pulleys 9 in relation to the traction sheave 18, the gripping angle A1 of the ropes around the traction sheave is set to a desired magnitude. In addition, the diverting pulleys 9 guide the rope sets going in opposite directions so that they run at equal distances from the guide rails 8. The centre line between the diverting pulleys 9 and that of the motor shaft lie substantially on the same straight line 7. The diverting pulleys 9 increase the frictional force between the rope 2 and the traction sheave 18 by increasing the angle of

contact A1 of the rope around the traction sheave, which is another advantage of the invention. Fig. 1 does not show the elevator guide rails and the supply of power to the electric equipment because these are outside the sphere of the invention.

The motor/counterweight of the invention can have a very flat construction. The width of the counterweight can be normal, i.e. somewhat narrower than the width of the elevator car. For an elevator designed for loads of about 800 kg, the diameter of the rotor of the motor of the invention is approx. 800 mm and the total counterweight thickness may be less than 160 mm. Thus, thanks to the flat motor construction, the counterweight of the invention can easily be accommodated in the space normally reserved for a counterweight. The large diameter of the motor provides the advantage that a gear is not necessarily needed. Placing the motor in the counterweight as provided by the invention allows the use of a larger motor diameter without involving any drawbacks. Although the motor has a larger diameter than a conventional motor, it can be easily fitted between the guide rails.

Fig. 2 presents section A-A of Fig. 1, showing the elevator motor 6. The motor 6 has a disc-shaped rotor 17 placed in the middle, so the motor has two air gaps in order to allow a higher torque. In this way, a symmetrical motor structure is achieved which is advantageous in respect of its strength properties, because the torsion applied to the traction sheave by the ropes is now transmitted to the motor shaft via a shorter lever arm. The motor 6 is placed at least partially inside the counterweight, and the motor is integrated with the counterweight 26 of the elevator by using at least one part of the motor, in this case an end shield, as a stator supporting element 11 which simultaneously forms a part of the counterweight, a side plate. Thus, the side plate 11 forms a frame part transmitting the load of the motor and counterweight. The structure comprises two side plates 11 and 12, with a shaft 13 between them. Attached to the side plate 11 is also the stator 14, with a stator winding 15 on it. Alternatively, the side plate 11 and the stator can be integrated in a single structure. Mounted on the shaft 13 by means of a bearing 16 is a disc-shaped rotor 17, which is placed substantially centrally in relation to the counterweight. The traction sheave 18 on the outer surface of the rotor 17 is provided with five rope grooves 19. The number of ropes may vary as required, but this embodiment has five ropes, each one of which makes an almost complete wind around the traction sheave. The traction sheave 18 can be a separate cylindrical body around the rotor 17, or the traction sheave and the rope grooves can be integrated with the rotor in a single body. The traction sheave is placed centrally in relation to the guide rails so that one half

of the rope sets 2a and 2b running in the same direction lies on one side of the plane 24 passing via the centre lines of the guide rails while the other half lies on the other side of said plane ( $a = b$ ). The rotor is provided with rotor windings 20, one on each side of the rotor disc (when a reluctance or synchronous motor is used, the rotor is naturally constructed according to the requirements of those motor types). There are two air gaps between the rotor 17 and stator 14. The shaft 13 is fixed to the stator, but it could alternatively be fixed to the rotor, in which case the bearing would be placed between the rotor 17 and side plate 11 or both side plates 11 and 12. Attached to the side plates of the counterweight are guides 25 designed to guide the counterweight movement between the guide rails 8. The guides also serve to transmit the supporting forces resulting from the operation of the motor to the guide rails. Side plate 12 acts as an additional reinforcement and stiffener for the counterweight/motor structure, because the horizontal shaft 13, the guides 25 and the diverting pulleys 9 guiding the ropes are attached to opposite points on the two side plates. Alternatively, auxiliary flanges could be used to attach the shaft 13 to the side plates, but this is not necessary for the description of the invention. Similarly, the stator core packets of stampings could be fastened to ringlike parts of the side plates 11 and 12 and these parts could then be fixed with bolts to appropriate points in the side plates. The motor placed in the counterweight is also provided with a brake 21. The brake is placed between the rotor 17 and the side plates 11 and 12. The rotor disc surface under the brake 21 can be provided with a separate braking surface.

Fig. 3 presents a motor placed at least partially inside the counterweight which is otherwise identical with the one in Fig. 2 except that the stator 14 and its core of stampings and winding 15 is now built in a disc placed substantially in the middle of the motor 6 in the direction of its shaft 16. The figure shows only one half of the motor as seen in section A-A of Fig. 1. The rotor 17 and its windings 20 are divided into two discs 17a and 17b placed on either side of the stator 14. The motor has two air gaps as was the case in the motor of Fig. 2. The motor 6 is provided with a cooling fan 22 built inside the shaft 13a. The fan receives air through holes 23 in the side plates 11 and 12 and blows it through the motor and further out through holes provided in the rotor discs 17a and 17b. This arrangement also provides the advantage that the traction sheave 18 and therefore the elevator ropes 2 in its grooves 19 are effectively cooled by the air flow at the same time. The common part integrated with the motor 6 and counterweight 26 is the motor shaft 13a, which is a structure connecting and

bracing the side plates 11 and 12 of the counterweight. The side plates 11 and 12 could as well be called end shields of the motor, although they are in a way outside the motor.

It is obvious to a person skilled in the art that different embodiments of the invention are not restricted to the example described above, but that they may instead be varied within the scope of the claims presented below. It is therefore obvious to the skilled person that it is inessential to the invention whether the counterweight is regarded as being integrated with the elevator motor or the elevator motor with the counterweight, because the outcome is the same and only the designations might be changed. It makes no difference to the invention if e.g. the side plates of the counterweight are designated as parts of the motor or as parts of the counterweight.

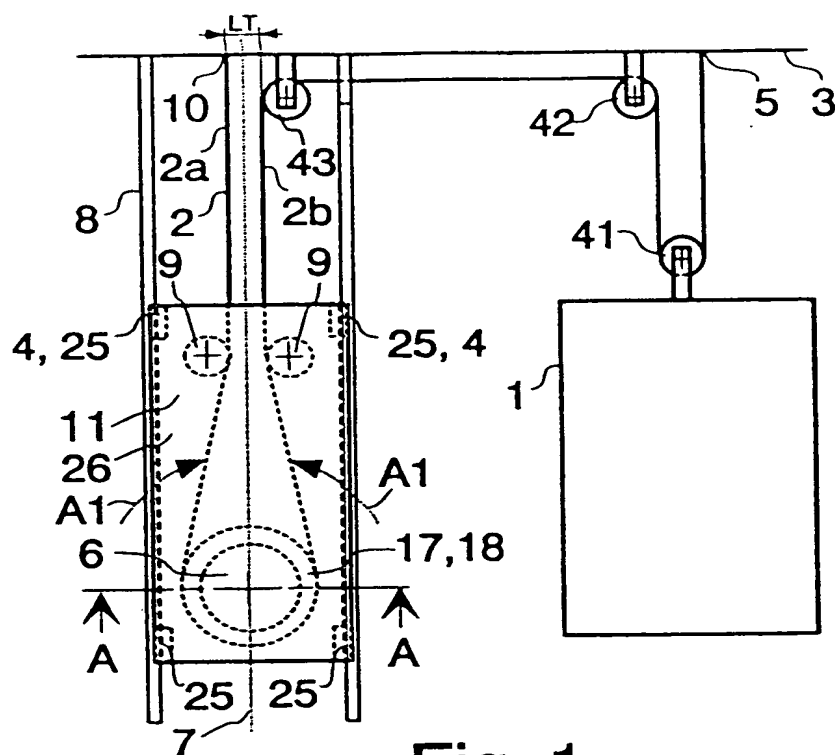
## Claims

1. Counterweight (26) of a rope-suspended elevator (1) moving along guide rails (8) and elevator machinery/motor (6) placed in the counterweight, said machinery/motor comprising a traction sheave (18), a bearing (16), a shaft (13), an element (11) supporting the bearing, a stator (14) provided with a winding (15) and a rotating, disc-shaped rotor (17), **characterized** in that the motor (6) of the elevator machinery has two stator windings (15), at least one disc-shaped rotor (17) and two rotor windings (20) and that the motor (6) has two air gaps ( ) between the rotor (17) and the stator (14), in which motor the planes formed by said air gaps are substantially perpendicular to the shaft (13) of the motor (6) and the traction sheave (18) is attached to the rotor.
2. Elevator machinery/motor (6) according to claim 1, **characterized** in that it has two disc-shaped rotors (17a,17b) and that the stator (14) is mounted between the rotors (17a,17b).
3. Elevator machinery/motor (6) according to claim 2, **characterized** in that the traction sheave (18) is mounted between the two rotors (17a,17b).
4. Elevator machinery/motor (6) according to any one of claims 1-3, **characterized** in that the elevator motor (6) is placed at least partially inside the counterweight (26).
5. Elevator machinery/motor (6) according to any one of claims 1-4, **characterized** in that the shaft (13) of the elevator motor (6) lies substantially on the centre line (7) between the

guide rails (8) of the counterweight (26).

6. Elevator machinery/motor (6) according to any one of claims 1-5, **characterized** in that at least one part of the elevator motor (6) is implemented as a common part with at least one structural part (11) or joined structural parts (11,13a) of the counterweight (26) of the elevator. 5
7. Elevator machinery/motor (6) according to claim 6, **characterized** in that the part of the elevator motor (6) which forms a structural part in common with the counterweight (26) is the element (11) supporting the stator (14) of the elevator motor, said element constituting a side plate (11) forming the frame of the counterweight (26). 10
8. Elevator machinery/motor (6) according to claim 7, **characterized** in that the stator (14,15) is fixedly connected to a side plate (11) forming the frame of the counterweight (26) and that the rotating rotor (17) provided with a traction sheave (18) is also connected to said side plate (11) via the bearing (16) and the shaft (13). 15
9. Elevator machinery/motor (6) according to claim 7 or 8, **characterized** in that the motor shaft (13) is fixed to the side plate (11) of the counterweight (26) and the bearing (16) is placed between the shaft (13) and the rotor (17). 20
10. Elevator machinery/motor (6) according to claim 8 or 9, **characterized** in that the shaft (13) is fixed to the rotor (17) and the bearing (16) is between the shaft (13) and the side plate (11). 25
11. Elevator machinery/motor (6) according to any one of claims 7-10, **characterized** in that the elevator motor (6) is provided with a brake (21) which is placed between the side plate (11) of the counterweight (26) or the stator (14) fixed to it and the rotor (17) or the shaft (13) fixed to it. 30
12. Elevator machinery/motor (6) according to any one of claims 9-11, **characterized** in that it has at least one diverting pulley (9) mounted on the side plate (11) forming the frame of the the counterweight, by means of which pulley (9) the contact angle (A1) of the rope (2) running around the traction sheave (18) is varied. 35

13. Elevator machinery/motor (6) according to any one of claims 1-12, **characterized** in that the counterweight (26) is provided with two diverting pulleys (9) between which the ropes (2) run and by means of which the contact angle (A1) of the rope (2) around the traction sheave (18) is varied, said diverting pulleys being so placed on the counterweight (26) that the midline between elevator ropes (2a,2b) going in different directions lies midway between the elevator guide rails and that the midline between elevator ropes (a,b) going in the same direction lies in the plane (24) passing through the centre lines of the guide rails (8). 40
14. Elevator machinery/motor (6) according to any one of claims 7-13, **characterized** in that, to guide the counterweight along the guide rails (8), the counterweight is provided with at least one guide (25), which is attached to the side plate (11) forming the frame of the counterweight. 45





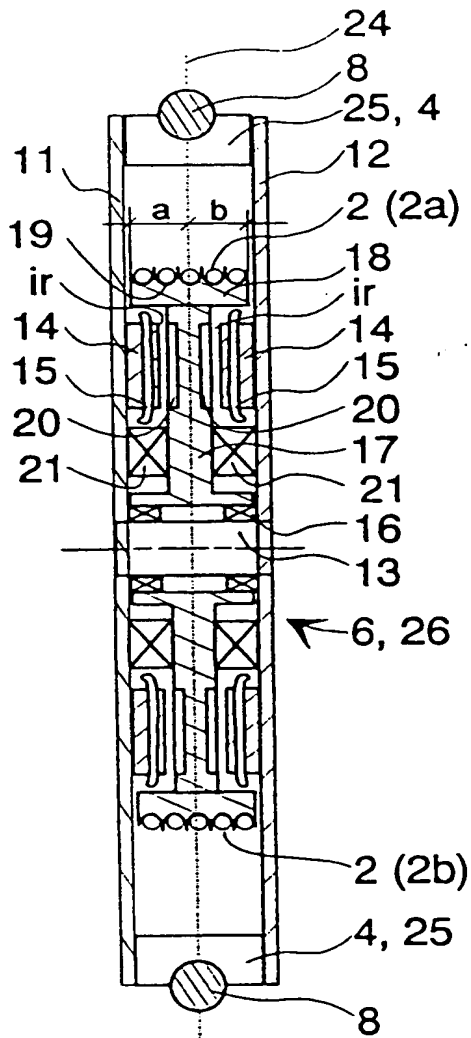


Fig. 2

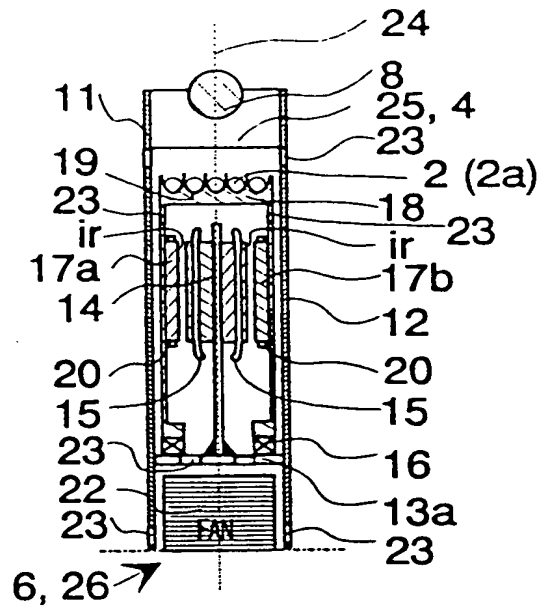


Fig. 3

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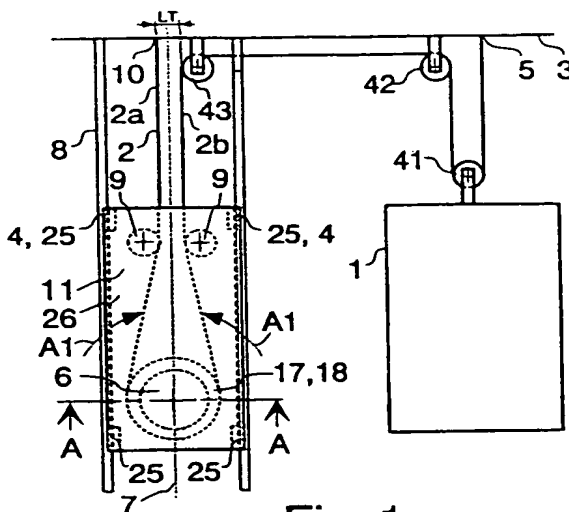
(11) Publication number:

**0 631 969 A3**

(12)

**EUROPEAN PATENT APPLICATION**(21) Application number: **94109886.5**(51) Int. Cl.<sup>6</sup>: **B66B 11/04**(22) Date of filing: **27.06.94**(30) Priority: **28.06.93 FI 932972**(43) Date of publication of application:  
**04.01.95 Bulletin 95/01**(84) Designated Contracting States:  
**AT BE CH DE DK ES FR GB GR IE IT LI LU MC  
NL PT SE**(88) Date of deferred publication of the search report:  
**03.05.95 Bulletin 95/18**(71) Applicant: **KONE OY**  
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**D-80639 München (DE)**(54) **Elevator drive machine placed in the counterweight.**

(57) In the present invention, a rotating elevator motor (6) provided with a traction sheave (18) is placed in the counterweight (26) of an elevator suspended with ropes (2). A gear system is not necessarily needed because, according to the invention, the structure and placement of the motor allow the use of a motor having a large diameter and a high torque. As the length of the motor still remains small, the motor/counterweight of the invention can be accommodated in the space normally reserved for a counterweight in the elevator shaft. The motor shaft lies in the counterweight (26) substantially midway between the guide rails (8) and the number (a,b) of ropes is equal on both sides of the plane (24) going through the centre lines of the guide rails.

**Fig. 1****EP 0 631 969 A3**



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## EUROPEAN SEARCH REPORT

Application Number  
EP 94 10 9886

DOCUMENTS CONSIDERED TO BE RELEVANT			
Category	Citation of document with indication, where appropriate, of relevant passages	Relevant to claim	CLASSIFICATION OF THE APPLICATION (Int.Cls)
A	US-A-4 960 186 (HONDA) * column 1, line 44 - line 49 * * column 2, line 3 - line 6 * * column 3, line 13 - line 40 * * column 3, line 48 - column 4, line 5 * * figures * ---	1,2	B66B11/04 B66B17/12
E	EP-A-0 630 849 (KONE OY)  * the whole document * -----	1-5,13, 14	
			TECHNICAL FIELDS SEARCHED (Int.Cl.5)
			B66B
The present search report has been drawn up for all claims			
Place of search THE HAGUE		Date of completion of the search 2 February 1995	Examiner Salvador, D
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